

ASME B1.1-2024
(Revision of ASME B1.1-2019)

Unified Inch Screw Threads (UN, UNR, and UNJ Thread Forms)

AN AMERICAN NATIONAL STANDARD



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Mechanical Engineers**

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**The American Society of
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FOREWORD

ASME B1.1, Unified Inch Screw Threads, is an integrated system of threads for fastening purposes in mechanisms and structures. Its outstanding characteristic is its general interchangeability of threads, achieved through the standardization of thread form, diameter-pitch combinations, and limits of size.

This Standard is the outgrowth of and supersedes previous editions that were published as ASME B1-1924, ASME B1.1-1935, ASME B1.1-1949, ASME B1.1-1960, ASME B1.1-1974, ASME B1.1-1982, ASME B1.1-1989, ASME B1.1-2003, and ASME B1.1-2019.

The achievements represented by ASME B1.1 in development, standardization, and unification are the result of the cooperation and coordination of many organizations, including The American Society of Mechanical Engineers (ASME), SAE International (formerly Society of Automotive Engineers), National Institute of Science and Technology (formerly National Bureau of Standards), Committee B1, the former National Screw Thread Commission, the former Interdepartmental Screw Thread Committee, British Standards Institution, CSA Group (formerly Canadian Standards Association), and American National Standards Institute (ANSI).

This Standard has its basis in the work done more than a century ago by William Sellers in the United States and Sir Joseph Whitworth in Great Britain. Through the intervening years, there have been many developments and revisions, culminating in the Unified Thread Standard approved and adopted for use by all inch-using countries.

The unification of screw thread standards meets the need for interchangeability among the billions of fasteners made in different countries and used in the complex equipment of modern technology. Unification is equally important for the international trade in mechanisms of all kinds and the servicing of transportation equipment that moves from country to country. Unification is therefore not only highly advantageous but also essential.

Complete unification of certain thread series and six tolerance classes in sizes $\frac{1}{4}$ in. and larger was achieved with the signing of an accord in Washington, D.C., on November 18, 1948. Since that time, unification has extended to smaller sizes.

Developed by Technical Committee No. 1 of the International Organization for Standardization (ISO), the unified inch standard that was adopted as ISO 5864 is parallel to the ISO metric screw thread system. Both systems have a common basic profile. The standard was subject to Quadripartite Standardization Agreement (QSTAG) 247 in the ABCA Army Standardization Program of America, Britain, Canada, and Australia.

Throughout this history, special attention has been given to the practical aspects of thread standardization, and many details of ASME B1.1 result from studies and tests based on real-world use. For example, users communicated the need for free assembly in high-production industries and the desirability of providing for threads that require a coating. The tolerance Classes 2A and 2B were developed to meet these two major requirements as well as to provide a general standard for externally and internally threaded fasteners. Thread symbols and nomenclature are now consistent with ASME B1.7. Thread acceptability now follows ASME B1.3.

In 1992, ASME B1.30 implemented eight-place decimal and rounding rules that are mandatory for all new editions and future revisions of ASME B1 documents. To comply with this decision, ASME B1.1-2003 revised some of the values in Table 2 and created [Nonmandatory Appendix E, Table E-1](#), which identifies and lists the revised dimensions from Table 2 in ASME B1.1-1989. The majority of the dimensional changes are within ± 0.0001 in. As stated in [para. 8.2.1](#), the values in this former Table 2, now [Tables 2A and 2B](#), and [Table E-1](#) should be considered acceptable until a future revision of this Standard makes the values in [Tables 2A and 2B](#) the only acceptable values.

ASME B1.1-1992 also moved Table 3B, which provided calculated values for various UNS (unified specials), to [Nonmandatory Appendix D](#). The ASME B1 Committee strongly urges users to adopt the standard thread sizes in [Tables 2A and 2B](#) instead of those listed in [Table D-1](#). ASME B1.1-1992 also moved Tables 31 through 40, which include some values that differed from those derived by use of the formulas in [sections 5 and 8](#), to [Nonmandatory Appendix D](#) and redesignated these Tables D-2 through D-11. (All future special threads should be based on calculations only.)

In addition, ASME B1.1-1992 eliminated all references to thread engagement. Past changes in the thread form designation of the “basic” thread height from $0.7500H$ to $0.62500H$ confused the calculation of percent of thread engagement.

The 1992 edition also included the definition of “functional diameter” and added the term to Table 2 in the same column as “pitch diameter,” since both characteristics have the same limits of size.

Finally, the 1992 edition explained in greater depth the effects of coating on threads (see [section 7](#)).

Changes to the 2019 edition included the splitting of ASME B1.1-2003, Table 2, which contained values for both internal and external threads for UN and UNR only, into two tables, [Table 2A](#), Limits of Size for Standard Series External Threads (UN, UNR, and UNJ), and [Table 2B](#), Limits of Size for Standard Series Internal Threads (UN, UNR, and UNJ). The metric translation of this Standard was removed, as were Tables D-2 through D-11.

The UNJ thread profile, formerly defined in ASME B1.15, was added to ASME B1.1-2019. Following the U.S. Department of Defense (DoD) approval of SAE AS8879C-2003, ASME B1 Subcommittee 15 recognized it would become the standard used by the aerospace industry for this thread form. As a result, Subcommittee 15 recommended that the technical information from ASME B1.15 be included in ASME B1.1 for nonaerospace applications.

The UNJ thread form having the enlarged root radius in the external thread was introduced to minimize size and weight in parts for applications requiring high-fatigue strength under high working-stress levels, as in aerospace applications. It is also appropriate for designs in commercial products where stresses are critical. To meet these requirements, the UNJ external thread root radius is designed to be between $0.15011107P$ and $0.18042196P$ and the minor diameter of the mating internal thread is increased to ensure the necessary clearance.

This Standard includes Classes 2A and 2B UNJ screw threads. Either Class 2A or Class 3A UNJ threads are appropriate for commercial applications commensurate with the fatigue and stress levels required.

The UNJ thread form is the UN thread form modified to $0.48713929P$ ($0.562500H$), which allows the $0.18042196P$ maximum root radius in the external thread. The first known U.S. standard of similar thread form was SAE AS82, published in March 1942, which describes a modified American National thread form to 75% h basic thread depth and specifies $0.10800P$ to $0.1800P$ root radius in the external thread. This thread was symbolized NR, National Round, and was developed for aircraft engine applications.

Tension fatigue testing of aircraft fasteners in 1942 demonstrated the importance of the external thread root contour in the fatigue life of a screw thread rolled after heat treatment. Fatigue testing isolated the elements of good external thread root design. The root should be radiused, not sharp. Theoretically, it should be a continuous circular arc, blending smoothly with the thread flanks. The radius should be as large as possible within the allowable design form. The root contour should also be smooth throughout and free of any imperfections, tool marks, or other minor notches.

Recognizing the need for improved 160,000 psi tensile strength bolts, the DoD published MIL-B-7838A, the bolt procurement specification for aircraft applications based on the unified thread form of $0.62500H$, in April 1952, thus acknowledging that a larger external root radius requires a shallower internal thread depth to clear the flank tangency point.

The root radius of the external thread was increased to $0.15011107P$ minimum and $0.18042196P$ maximum for the 180,000 psi and higher tensile strength bolts. This external thread form was developed in 1955 by the aerospace fastener industry and was known as the “Hi R” thread form.

Through coordinated effort with the SAE E-25 Engine and Propeller Standard Utility Parts Committee and the Aerospace Industries Association National Aerospace Standards Committee (NAS), the DoD developed and published in September 1960 the thread specification MIL-S-8879, which features the “Hi R” thread root radius in the external thread and the internal thread modified to $0.48713929P$ ($0.562500H$) basic. In aircraft gas turbine engines, the high-temperature threaded fasteners exhibited better elevated temperature performance using MIL-S-8879 UNJ thread root radius, as the stress-rupture life of bolts was greatly improved.

The UNJ thread form has been adopted by the aerospace industry as the all-purpose thread standard, except for electrical hardware and thread sizes 0.1380 and smaller, which may use the UN thread form.

The UNJ profile as defined in this Standard is similar to SAE AS8879C-2003 (superseding MIL-S-8879C) and equivalent to ISO 3161:1977 for thread Classes 3A and 3B. British Standards Institution BS 4084:1978, including Amendment 1, is technically identical to ISO 3161:1977, except for Appendix A, which provides information for a 20-UNJ constant pitch series for diameters through 3 in.

Changes to ASME B1.1-2024 include the following:

- (a) correction of TD_2 to Td_2 in four places.
- (b) correction of six minimum PD values in [Table 2A](#) by 0.0001.
- (c) changing values of UNR max. minor diameter in [Table 2A](#), column 13, due to a change in the $2h_s$ constant from 1.19078493 P to 1.22686932 P .
- (d) correction to the formula at the bottom of [Tables 6](#) through [16](#) for the area at the minor diameter. Values in the applicable column in the tables were correct; the formula was written incorrectly. Therefore, $\pi[(D - 2h_b/2)^2]$ is corrected to $\pi\{[(D - 2h_b)/2]^2\}$.
- (e) revision of three notes pertaining to UNR in [Figure 8](#), for better explanation and clarification.
- (f) changing the wording of [para. 5.2](#) so the length of engagement to be used is now defined. Previously, there was no length of engagement stated for the calculation of Classes 1A and 1B for 4-UN, 6-UN, and 8-UN threads.
- (g) addition of a dotted line to [Figure 12](#), so “ D min. = D bsc” is not interpreted as requiring a radius.

(h) the following corrections to [Table 18A](#):

(1) TD_2 to Td_2 in one place

(2) use of four decimal places rather than six in line (4)

(3) correction of “0.5000-23” to “0.5000-28” in table header

(i) deletion of references to H in [section 10](#). ASME B1 volumes stopped using references to H in 2003. If the equivalent H values are needed, they can be found in [Table 5](#). Other errors related to the use of H , including wrong values, incorrect references, and formulas written incorrectly, are also corrected in [section 10](#).

(j) revisions in [para. 10\(f\)](#), of UNR h_s from $0.59539247P$ to $0.61343466P$ and in [para. 10\(r\)](#) of UNR $2h_s$ from $1.19078493P$ to $1.22686932P$.

ASME B1.1-2024 was approved by ANSI on May 7, 2024.

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Standardization and Unification of Screw Threads

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Revisions and Errata. The committee processes revisions to this Standard on a continuous basis to incorporate changes that appear necessary or desirable as demonstrated by the experience gained from the application of the Standard. Approved revisions will be published in the next edition of the Standard.

In addition, the committee may post errata on the committee web page. Errata become effective on the date posted. Users can register on the committee web page to receive e-mail notifications of posted errata.

This Standard is always open for comment, and the committee welcomes proposals for revisions. Such proposals should be as specific as possible, citing the paragraph number, the proposed wording, and a detailed description of the reasons for the proposal, including any pertinent background information and supporting documentation.

Cases

(a) The most common applications for cases are

(1) to permit early implementation of a revision based on an urgent need

(2) to provide alternative requirements

(3) to allow users to gain experience with alternative or potential additional requirements prior to incorporation directly into the Standard

(4) to permit the use of a new material or process

(b) Users are cautioned that not all jurisdictions or owners automatically accept cases. Cases are not to be considered as approving, recommending, certifying, or endorsing any proprietary or specific design, or as limiting in any way the freedom of manufacturers, constructors, or owners to choose any method of design or any form of construction that conforms to the Standard.

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(1) a statement of need and background information

(2) the urgency of the case (e.g., the case concerns a project that is underway or imminent)

(3) the Standard and the paragraph, figure, or table number

(4) the editions of the Standard to which the proposed case applies

(d) A case is effective for use when the public review process has been completed and it is approved by the cognizant supervisory board. Approved cases are posted on the committee web page.

Interpretations. The committee does not issue interpretations for this Standard.

Committee Meetings. The B1 Standards Committee regularly holds meetings that are open to the public. Persons wishing to attend any meeting should contact the secretary of the committee. Information on future committee meetings can be found on the committee web page at <https://go.asme.org/B1committee>.

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UNIFIED INCH SCREW THREADS (UN, UNR, AND UNJ THREAD FORMS)

1 GENERAL

1.1 Scope

This Standard specifies the thread form, series, class, allowance, tolerance, and designation for unified screw threads. (In order to emphasize that unified screw threads are based on inch modules, they may be denoted unified inch screw threads.) Several variations in thread form have been developed for unified threads; however, this Standard covers only UN, UNR, and UNJ thread forms.

The metric translation of this Standard that was in the 2003 edition has been removed (see [Nonmandatory Appendix C](#)). [Nonmandatory Appendices D](#) through [F](#) contain information that is supplementary to the sections of this Standard.

1.2 Unified Screw Thread Standards

The standards for unified screw threads published in this Standard are in agreement with formal standards of the International Organization for Standardization (ISO) for diameter-pitch combinations, designations, and tolerances for 60-deg triangular form inch screw threads. The unified screw thread symbols UN, UNC, UNF, and UNEF were derived by the addition of the letter “U” preceding the thread symbols used for American National screw threads N, NC, NF, and NEF.

Unified screw threads have their origin in an accord signed in Washington, D.C., on November 18, 1948, by representatives of standardizing bodies of Canada, the United Kingdom, and the United States and have subsequently superseded American National screw threads.

1.3 Thread Forms

UN applies to both internal and external threads. UNR applies only to external threads; the difference between UN and UNR threads, in addition to designation, is that a flat or rounded root contour due to tool wear is specified for UN threads, while only a defined rounded root contour is specified for UNR threads. Basic thread height is 0.54126588*P*.

The UNJ screw thread is designed for use on highly stressed applications requiring high-fatigue strength. For aerospace applications, only Classes 3A and 3B should be used. Basic thread height is only

0.48713929*P* to permit a root radius larger than that of the UN and UNR forms.

1.4 Interchangeability

1.4.1 UN and UNR. Unified (UN/UNR) and its predecessor American National (N) screw threads have substantially the same thread form, and threads of both standards having the same diameter and pitch are mechanically interchangeable. The principal differences between these standards relate to the application of allowances, the variation of tolerances with size, differences in the amounts of pitch diameter tolerances for external and internal threads, and differences in thread designations. Unified inch and ISO metric screw threads are not mechanically interchangeable.

1.4.2 UNJ. UN and UNJ threads are interchangeable with the exception of UNJ-3A external threads, which at maximum material condition will not assemble with a UN internal thread of any class at maximum material condition (see [Figure 1](#)).

1.5 Designations

Unified thread sizes (specific combinations of diameter and pitch shown in [Table 1](#)) are identified by the letter combination “UN” in the thread symbol. In the unified standards, the pitch diameter tolerances for external threads differ from those for internal threads; for this reason the letter “A” is used in the thread symbol to denote an external thread and the letter “B,” an internal thread. Where the letters “U,” “A,” or “B” do not appear in the thread designation, the threads conform to the outdated American National screw threads. Details regarding thread designations are given in [section 6](#).

1.6 References

The following is a list of publications referenced in this Standard. Unless otherwise specified, the latest edition shall apply. The following documents form a part of this Standard to the extent specified herein.

ASME B1.2. Gages and Gaging for Unified Inch Screw Threads. The American Society of Mechanical Engineers.